

# SimpleFEM

an introduction to the Q1P0 element and its implementation

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## Warning



The following presentation is about the Finite Element Method but it isn't

- ▶ comprehensive

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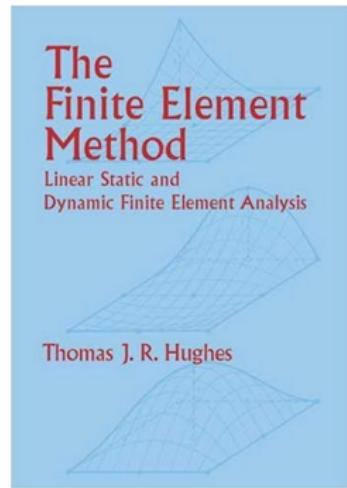
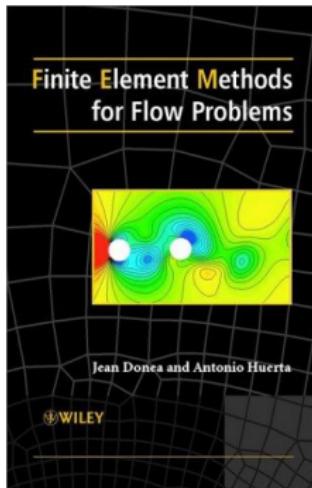
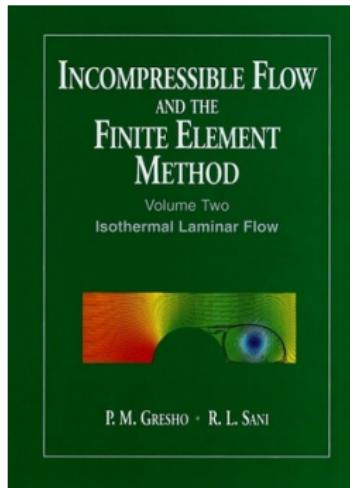


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→ focus on incompressible Stokes flow, Q1P0 element

# Literature



## The physics

The mechanical behavior of Earth materials which compose the crust and the mantle is described by means of the Stokes equation :

$$\nabla \cdot \sigma + \mathbf{b} = 0$$

Solenoidal constraint due to the **incompressibility** condition :

$$\nabla \cdot \mathbf{v} = 0$$

In order for the problem to be closed, the stress tensor must be related to velocity and pressure :

$$\sigma = -p\mathbf{1} + \mathbf{s}$$

The deviatoric stress tensor is related to the velocity gradient through the dynamic viscosity  $\mu$  as follows :

$$\mathbf{s} = 2\mu\dot{\epsilon} \quad \dot{\epsilon} = \frac{1}{2} \left( \nabla \mathbf{v} + (\nabla \mathbf{v})^T \right)$$

## Penalty formulation

Material is assumed to be weakly compressible :

$$p = -\lambda \nabla \cdot \mathbf{v} = -\lambda(\dot{\epsilon}_{xx} + \dot{\epsilon}_{yy})$$

$\lambda$  is the penalty factor ( $\sim$  bulk viscosity), with  $\lambda \gg \mu$ .

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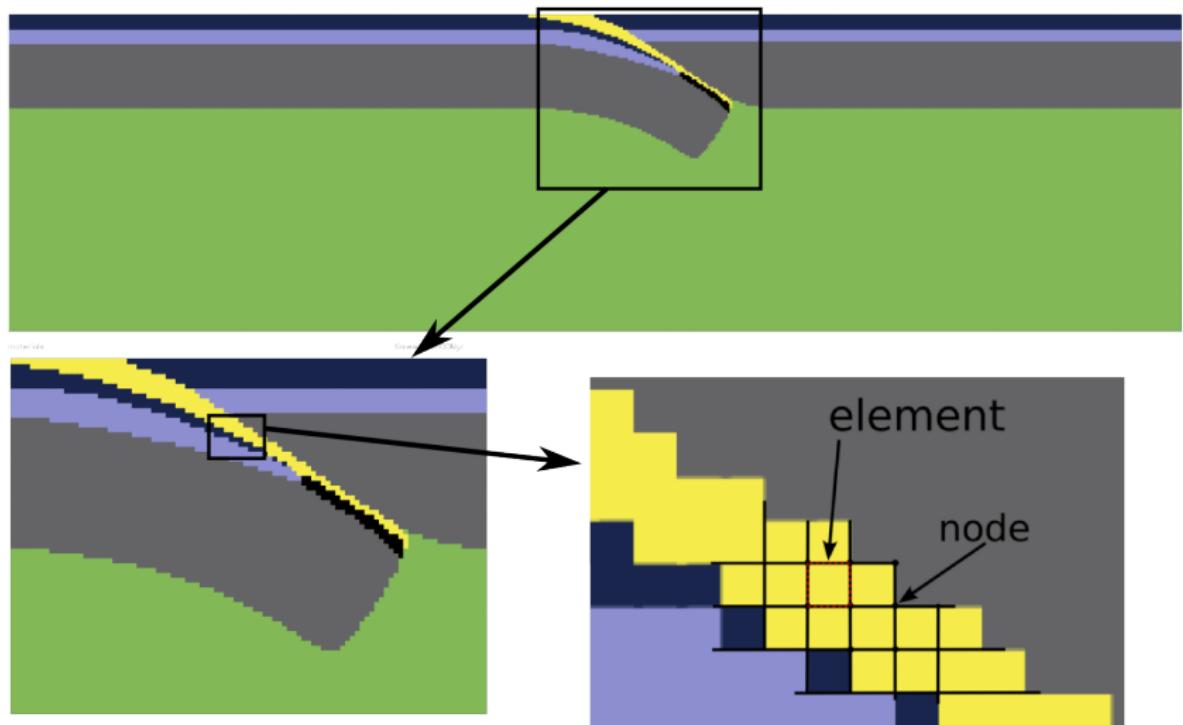
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$$\left. \begin{array}{l} \nabla \cdot \boldsymbol{\sigma} + \mathbf{b} = \mathbf{0} \\ \boldsymbol{\sigma} = -p\mathbf{1} + \mathbf{s} \\ \mathbf{s} = 2\mu\dot{\boldsymbol{\epsilon}} \\ \dot{\boldsymbol{\epsilon}} = \frac{1}{2} (\nabla \mathbf{v} + (\nabla \mathbf{v})^T) \\ p = -\lambda \cdot \nabla \cdot \mathbf{v} \end{array} \right\} \Rightarrow \nabla \cdot (\mu \nabla \mathbf{v}) + \lambda \nabla (\nabla \cdot \mathbf{v}) + \mathbf{b} = 0$$

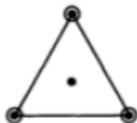
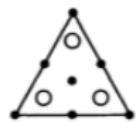
$\Rightarrow$  decoupling of  $\mathbf{v}$  and  $p$  equations.

## Concept



PDE's → discretisation → FEM formulation →  $\mathbf{A} \cdot \mathbf{x} = \mathbf{B}$

# Which element ?



**Crouzeix–Raviart element:**  
Velocity: continuous quadratic  
+ cubic bubble function,  
Pressure: discontinuous linear,  
Satisfies LBB condition,  
Quadratic convergence.

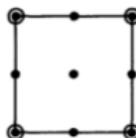
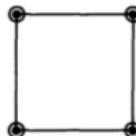
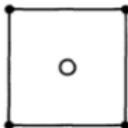
**Mini element:**  
Velocity: continuous linear  
+ cubic bubble function,  
Pressure: continuous linear,  
Satisfies LBB condition,  
Linear convergence.

**Nodes:** • Velocity  
○ Pressure

(Donea & Huerta)



Not all elements are born equal.



**Q1P0 element:**  
Continuous bilinear velocity,  
Discontinuous constant pressure,  
Does not satisfy LBB condition.  
(same for linear/constant triangle)

**Q1Q1 element:**  
Continuous bilinear velocity,  
Continuous bilinear pressure,  
Does not satisfy LBB condition.  
(same for linear/linear triangle)

**Q2Q1 element:**  
(Taylor–Hood element)  
Continuous biquadratic velocity,  
Continuous bilinear pressure,  
Satisfies LBB condition,  
Quadratic convergence.  
(same for quadratic/linear triangle)

# Which element ?

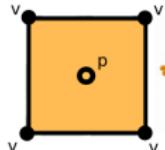


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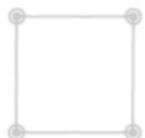
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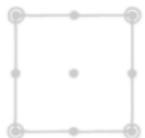
**Q1P0 element**

**Continuous bilinear velocity**  
**Discontinuous constant pressure**



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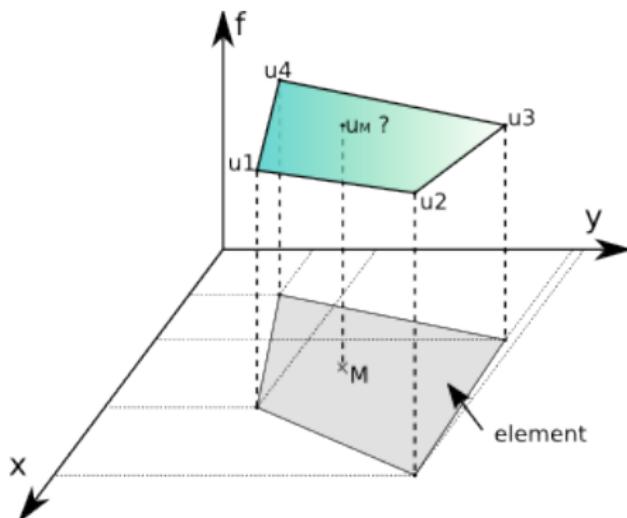
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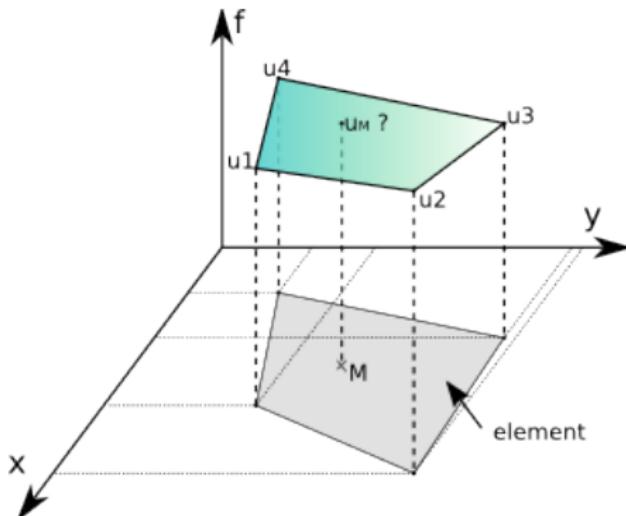
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## Shape functions



$$u_M = f(u_1, u_2, u_3, u_4, x, y)$$

## Shape functions



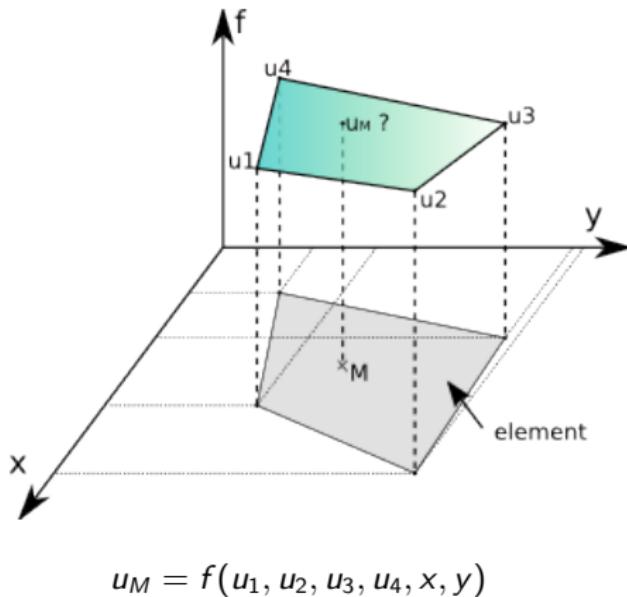
$$u_M = f(u_1, u_2, u_3, u_4, x, y)$$

For  $M(x,y)$  inside the element :

$$u(x, y) = \sum_{i=1}^4 N_i(x, y) u_i$$

$$v(x, y) = \sum_{i=1}^4 N_i(x, y) v_i$$

## Shape functions



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$$v(x,y) = \sum_{i=1}^4 N_i(x,y) v_i$$

$$\dot{\epsilon}_{xx}(x,y) = \frac{\partial u}{\partial x} = \sum_{i=1}^4 \frac{\partial N_i}{\partial x} u_i$$

$$\dot{\epsilon}_{yy}(x,y) = \frac{\partial v}{\partial y} = \sum_{i=1}^4 \frac{\partial N_i}{\partial y} v_i$$

## under the bonnet of the FEM engine (1)

The tensor  $\sigma$  is symmetric (*i.e.*  $\sigma_{xy} = \sigma_{yx}$ ).

It can be cast in vector format :

$$\begin{pmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{pmatrix} = \begin{pmatrix} -p \\ -p \\ 0 \end{pmatrix} + 2\mu \begin{pmatrix} \dot{\epsilon}_{xx} \\ \dot{\epsilon}_{yy} \\ \dot{\epsilon}_{xy} \end{pmatrix}$$

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$$= \lambda \begin{pmatrix} \dot{\epsilon}_{xx} + \dot{\epsilon}_{yy} \\ \dot{\epsilon}_{xx} + \dot{\epsilon}_{yy} \\ 0 \end{pmatrix} + 2\mu \begin{pmatrix} \dot{\epsilon}_{xx} \\ \dot{\epsilon}_{yy} \\ \dot{\epsilon}_{xy} \end{pmatrix}$$

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## under the bonnet of the FEM engine (2)

Remember that

$$\frac{\partial u}{\partial x} = \sum_{i=1}^4 \frac{\partial N_i}{\partial x} u_i \quad \frac{\partial v}{\partial y} = \sum_{i=1}^4 \frac{\partial N_i}{\partial y} v_i$$

$$\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = \sum_{i=1}^4 \frac{\partial N_i}{\partial y} u_i + \sum_{i=1}^4 \frac{\partial N_i}{\partial x} v_i$$

so that

$$\begin{pmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial v}{\partial y} \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{pmatrix} = \underbrace{\begin{pmatrix} \frac{\partial N_1}{\partial x} & 0 & \frac{\partial N_2}{\partial x} & 0 & \frac{\partial N_3}{\partial x} & 0 & \frac{\partial N_4}{\partial x} & 0 \\ 0 & \frac{\partial N_1}{\partial y} & 0 & \frac{\partial N_2}{\partial y} & 0 & \frac{\partial N_3}{\partial y} & 0 & \frac{\partial N_4}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_1}{\partial x} & \frac{\partial N_2}{\partial y} & \frac{\partial N_2}{\partial x} & \frac{\partial N_3}{\partial y} & \frac{\partial N_3}{\partial x} & \frac{\partial N_4}{\partial y} & \frac{\partial N_4}{\partial x} \end{pmatrix}}_B \cdot \begin{pmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \\ u_4 \\ v_4 \end{pmatrix}_v.$$

## under the bonnet of the FEM engine (3)

Finally,

$$\begin{pmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{pmatrix} = (\lambda \mathbf{K} + \mu \mathbf{C}) \cdot \mathbf{B} \cdot \mathbf{V}$$

## Weak form (1)

We start again from

$$\nabla \cdot \sigma + \mathbf{b} = 0$$

For the  $N_i$ 's 'regular enough', we can write :

$$\int_{\Omega_e} N_i \nabla \cdot \sigma d\Omega + \int_{\Omega_e} N_i \mathbf{b} d\Omega = 0$$

We can integrate by parts and drop the surface term :

$$\int_{\Omega_e} \nabla N_i \cdot \sigma d\Omega = \int_{\Omega_e} N_i \mathbf{b} d\Omega$$

or,

$$\int_{\Omega_e} \begin{pmatrix} \frac{\partial N_i}{\partial x} & 0 & \frac{\partial N_i}{\partial y} \\ 0 & \frac{\partial N_i}{\partial y} & \frac{\partial N_i}{\partial x} \end{pmatrix} \cdot \begin{pmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{pmatrix} d\Omega = \int_{\Omega_e} N_i \mathbf{b} d\Omega$$

## Weak form (2)

Let  $i = 1, 2, 3, 4$  and let us define

$$\mathbf{N}_b^T = (N_1 b_x, N_1 b_y, \dots, N_4 b_x, N_4 b_y)$$

then we can write

$$\int_{\Omega_e} \mathbf{B}^T \cdot [\lambda \mathbf{K} + \mu \mathbf{C}] \cdot \mathbf{B} \cdot \mathbf{V} d\Omega = \int_{\Omega_e} \mathbf{N}_b d\Omega$$

and finally :

$$\underbrace{\left( \int_{\Omega_e} \mathbf{B}^T \cdot [\lambda \mathbf{K} + \mu \mathbf{C}] \cdot \mathbf{B} d\Omega \right)}_{A_{el}(8 \times 8)} \cdot \underbrace{\mathbf{V}}_{(8 \times 1)} = \underbrace{\int_{\Omega_e} \mathbf{N}_b d\Omega}_{B_{el}(8 \times 1)}$$

or,

$$\left[ \underbrace{\left( \int_{\Omega_e} \lambda \mathbf{B}^T \cdot \mathbf{K} \cdot \mathbf{B} d\Omega \right)}_{A_{el}^\lambda(8 \times 8)} + \underbrace{\left( \int_{\Omega_e} \mu \mathbf{B}^T \cdot \mathbf{C} \cdot \mathbf{B} d\Omega \right)}_{A_{el}^\mu(8 \times 8)} \right] \cdot \underbrace{\mathbf{V}}_{(8 \times 1)} = \underbrace{\int_{\Omega_e} \mathbf{N}_b d\Omega}_{B_{el}(8 \times 1)}$$

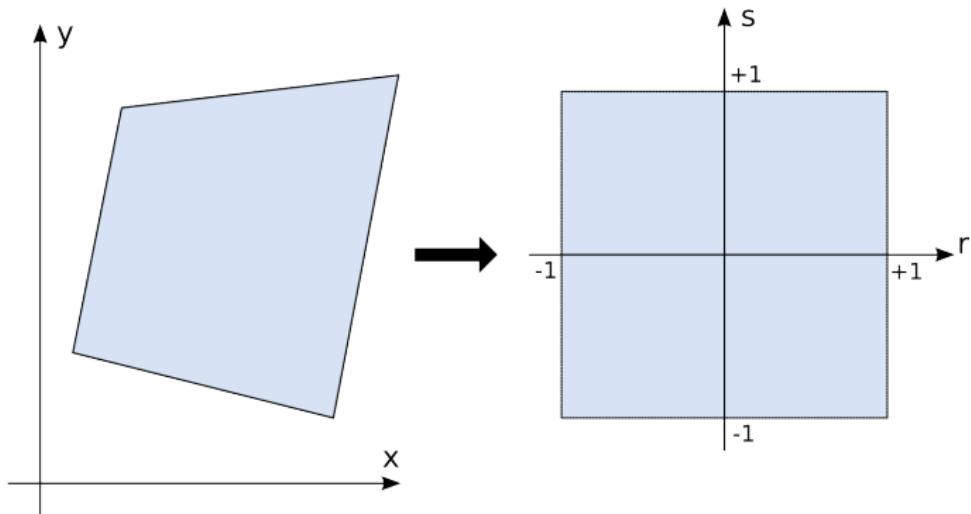
## Numerical integration (1)

We need to compute quantities involving integrals over the element.  
It is carried out in two steps :

1. change of variables
2. quadrature

## Numerical integration (2)

To exploit the full flexibility of FEM, we employ isoparametric elements. Each element in physical space is mapped onto the reference element with fixed shape, size, and orientation.



## Numerical integration (3)

Let  $f$  be a function. From the chain rule of calculus, we write

$$\frac{\partial f}{\partial r} = \frac{\partial f}{\partial x} \frac{\partial x}{\partial r} + \frac{\partial f}{\partial y} \frac{\partial y}{\partial r}$$

$$\frac{\partial f}{\partial s} = \frac{\partial f}{\partial x} \frac{\partial x}{\partial s} + \frac{\partial f}{\partial y} \frac{\partial y}{\partial s}$$

or in matrix form :

$$\begin{pmatrix} \frac{\partial f}{\partial r} \\ \frac{\partial f}{\partial s} \end{pmatrix} = \underbrace{\begin{pmatrix} \frac{\partial x}{\partial r} & \frac{\partial y}{\partial r} \\ \frac{\partial x}{\partial s} & \frac{\partial y}{\partial s} \end{pmatrix}}_{\mathbf{J}} \cdot \begin{pmatrix} \frac{\partial f}{\partial x} \\ \frac{\partial f}{\partial y} \end{pmatrix}$$

where  $\mathbf{J}$  is called the Jacobian of the transformation

## Numerical integration (4)

We have :

$$\begin{pmatrix} \frac{\partial f}{\partial x} \\ \frac{\partial f}{\partial y} \end{pmatrix} = \mathbf{J}^{-1} \cdot \begin{pmatrix} \frac{\partial f}{\partial r} \\ \frac{\partial f}{\partial s} \end{pmatrix}$$

One can prove that

$$\int \int_{\Omega_e} dx dy = \int_{-1}^{+1} \int_{-1}^{+1} |J| dr ds$$

so that for instance

$$\int \int_{\Omega_e} \frac{\partial f}{\partial x} dx dy = \int_{-1}^{+1} \int_{-1}^{+1} \left( \tilde{J}_{11} \frac{\partial f}{\partial r} + \tilde{J}_{12} \frac{\partial f}{\partial s} \right) |J| dr ds$$

where  $\tilde{J}_{ij}$  are the components of  $\mathbf{J}^{-1}$ .

## Numerical integration (5) - the Gauss quadrature

- ▶ a quadrature rule is an approximation of the definite integral of a function, usually stated as a weighted sum of function values at specified points within the domain of integration.
- ▶ an  $n$ -point Gaussian quadrature rule is constructed to yield an exact result for polynomials of degree  $2n-1$

$$\int_{-1}^{+1} f(x) dx \approx \sum_{i=1}^n w_i f(x_i)$$

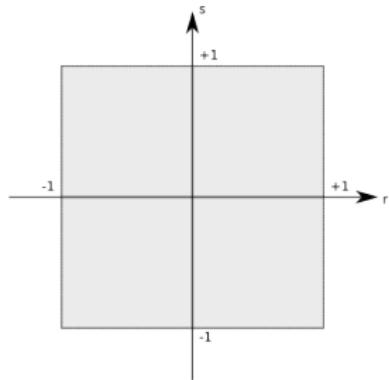
$$\int \int_{-1}^{+1} f(x, y) dx dy \approx \sum_{i=1}^n \sum_{j=1}^n w_i w_j f(x_i, y_j)$$

$$\Rightarrow \int_{\Omega_e} \lambda \mathbf{B}^T \cdot \mathbf{K} \cdot \mathbf{B} d\Omega = \sum_{i=1}^n \sum_{j=1}^n w_i w_j |J| \left( \lambda \mathbf{B}^T \cdot \mathbf{K} \cdot \mathbf{B} \right)_{(x_i, y_j)}$$

## Numerical integration (6) - the Gauss quadrature

number of points	position $r_i$	weight $w_i$
1	0	2
2	$-1/\sqrt{3}$	1
	$+1/\sqrt{3}$	1
3	$-\sqrt{15}/5$	$5/9$
	0	$8/9$
	$+\sqrt{15}/5$	$5/9$

## Shape functions (2)



$$\begin{aligned}N_1(r, s) &= 0.25(1 - r)(1 - s) \\N_2(r, s) &= 0.25(1 + r)(1 - s) \\N_3(r, s) &= 0.25(1 + r)(1 + s) \\N_4(r, s) &= 0.25(1 - r)(1 + s)\end{aligned}$$

$$\begin{aligned}\frac{\partial N_1}{\partial r}(r, s) &= -0.25(1 - s) \\ \frac{\partial N_2}{\partial r}(r, s) &= +0.25(1 - s) \\ \frac{\partial N_3}{\partial r}(r, s) &= +0.25(1 + s) \\ \frac{\partial N_4}{\partial r}(r, s) &= -0.25(1 + s)\end{aligned}$$

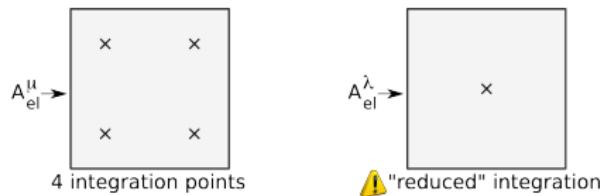
$$\begin{aligned}\frac{\partial N_1}{\partial s}(r, s) &= -0.25(1 - r) \\ \frac{\partial N_2}{\partial s}(r, s) &= -0.25(1 + r) \\ \frac{\partial N_3}{\partial s}(r, s) &= +0.25(1 + r) \\ \frac{\partial N_4}{\partial s}(r, s) &= +0.25(1 - r)\end{aligned}$$

# FEM algorithm

1. partition domain  $\Omega$  into elements  $\Omega_e$ ,  $e = 1, \dots n_{el}$ .

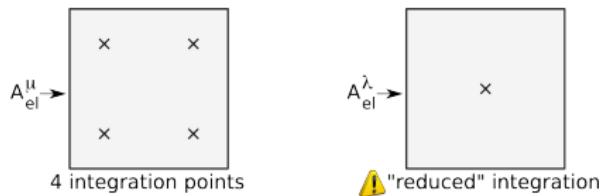
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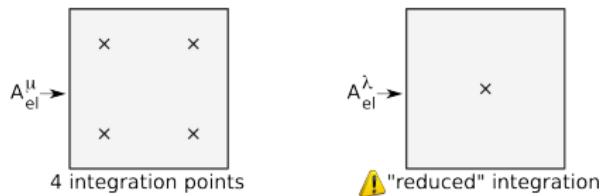
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3. a node belongs to several elements  
→ need to assemble  $\mathbf{A}_{el}$  and  $\mathbf{B}_{el}$  in  $\mathbf{A}$ ,  $\mathbf{B}$

# FEM algorithm

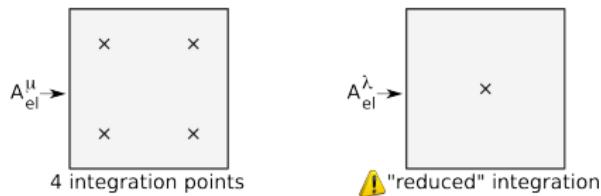
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4. apply boundary conditions

# FEM algorithm

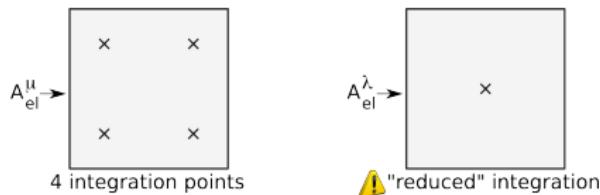
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- apply boundary conditions
- solve system :  $\mathbf{x} = \mathbf{A}^{-1} \cdot \mathbf{B}$

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- apply boundary conditions
- solve system :  $\mathbf{x} = \mathbf{A}^{-1} \cdot \mathbf{B}$
- visualise/analyse  $\mathbf{x}$

# SimpleFEM



- ▶ 2D
- ▶ fortran90
- ▶ simple (naive ?) approach
  - ▶ no attention to performance
  - ▶ no attention to memory
  - ▶ no subroutines, modules, ...
- ▶ available at : <http://cedricthieulot.net>

## Files

- ▶ `blas_routines.f90`  
contains a few subroutines from the BLAS (Basic Linear Algebra Subprograms) library
- ▶ `linpack_d.f90`  
software library for performing numerical linear algebra. makes use of the BLAS libraries for performing basic vector and matrix operations.
- ▶ `simplefem.f90`  
the FEM code

## Declarations (1)

terminology : "dof" = degree of freedom = unknown

```
integer,parameter :: m=4          ! number of nodes making an element
integer,parameter :: ndof=2        ! number of dofs per node

integer nnx                      ! number of grid points in the x direction
integer nny                      ! number of grid points in the y direction
integer np                        ! number of grid points

integer nelx                     ! number of elements in the x direction
integer nely                     ! number of elements in the y direction
integer nel                        ! number of elements

integer Nfem                      ! size of the FEM matrix

integer,dimension(:,:),allocatable:: icon ! connectivity array

integer,dimension(:),allocatable:: ipvt ! work array needed by the solver
```

## Declarations (2)

```
real(8) Lx,Ly           ! size of the numerical domain
real(8) viscosity        ! dynamic viscosity of the material
real(8) density          ! mass density of the material
real(8) gx,gy            ! gravity acceleration
real(8) penalty           ! penalty parameter lambda

real(8), dimension(:), allocatable :: x,y      ! node coordinates arrays
real(8), dimension(:), allocatable :: u,v      ! node velocity arrays
real(8), dimension(:), allocatable :: press     ! pressure

real(8), dimension(:), allocatable :: B          ! right hand side
real(8), dimension(:, :, :), allocatable :: A      ! FEM matrix

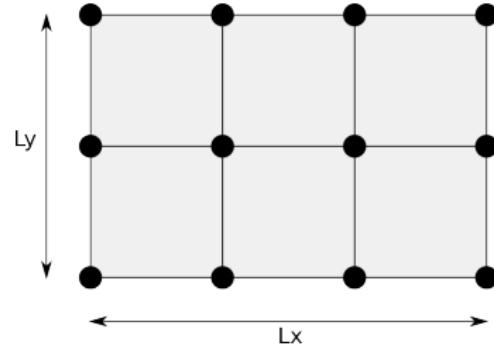
real(8), dimension(:), allocatable :: work       ! work array needed by the solver
real(8), dimension(:), allocatable :: bc_val     ! array containing bc values
```

## Declarations (3)

```
real(8), external :: b1,b2,uth,vth,pth
real(8) rq,sq,wq
real(8) xq,yq
real(8) uq,vq
real(8) exxq,eyyq,exyq
real(8) Ael(m*ndof,m*ndof)
real(8) Bel(m*ndof)
real(8) N(m),dNdx(m),dNdy(m),dNdr(m),dNds(m)
real(8) jacob
real(8) jcb(2,2)
real(8) jcobi(2,2)
real(8) Bmat(3,ndof*m)
real(8), dimension(3,3) :: Kmat
real(8), dimension(3,3) :: Cmat
real(8) Aref
real(8) eps
real(8) rcond
logical, dimension(:, ), allocatable :: bc_fix
logical, dimension(:, :, ), allocatable :: C
! body force and analytical solution
! local coordinate and weight of qpoint
! global coordinate of qpoint
! velocity at qpoint
! strain-rate components at qpoint
! elemental FEM matrix
! elemental right hand side
! shape fcts and derivatives
! determinant of jacobian matrix
! jacobian matrix
! inverse of jacobian matrix
! B matrix
! K matrix
! C matrix
!
!
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!
!
```

# Declarations

```
Lx=3.d0  
Ly=2.d0  
  
nnx=4  
nny=3  
  
np=nnx*nny  
  
nelx=nnx-1  
nely=nny-1  
  
nel=nelx*nely  
  
viscosity=1.d0  
density=1.d0  
  
penalty=1.d7  
  
Nfem=np*ndof  
  
eps=1.d-10  
  
Kmat(1,1)=1.d0;Kmat(1,2)=1.d0;Kmat(1,3)=0.d0  
Kmat(2,1)=1.d0;Kmat(2,2)=1.d0;Kmat(2,3)=0.d0  
Kmat(3,1)=0.d0;Kmat(3,2)=0.d0;Kmat(3,3)=0.d0  
  
Cmat(1,1)=2.d0;Cmat(1,2)=0.d0;Cmat(1,3)=0.d0  
Cmat(2,1)=0.d0;Cmat(2,2)=2.d0;Cmat(2,3)=0.d0  
Cmat(3,1)=0.d0;Cmat(3,2)=0.d0;Cmat(3,3)=1.d0
```



$$\mathbf{K} = \begin{pmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 0 \end{pmatrix}$$
$$\mathbf{C} = \begin{pmatrix} 2 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

# Allocating memory

```
allocate(x(np))
allocate(y(np))
allocate(u(np))
allocate(v(np))

allocate(press(nel))

allocate(icon(m, nel))

allocate(A(Nfem, Nfem))
allocate(B(Nfem))
allocate(C(Nfem, Nfem))

allocate(bc_fix(Nfem))
allocate(bc_val(Nfem))
```

## Allocating memory

```
allocate(x(np))
allocate(y(np))
allocate(u(np))
allocate(v(np))

allocate(press(nel))

allocate(icon(m, nel))

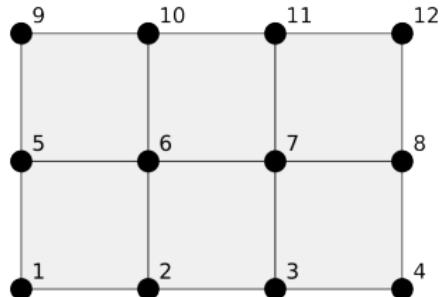
allocate(A(Nfem, Nfem))
allocate(B(Nfem))
allocate(C(Nfem, Nfem))

allocate(bc_fix(Nfem))
allocate(bc_val(Nfem))
```

- ▶  $\text{size}(A) = Nfem^2 = ndof^2 \times np^2$
- ▶ with  $ndof=2$ , in double precision :
  - $np=10^2 \quad \text{size}(A) = 320\text{kb}$
  - $np=50^2 \quad \text{size}(A) = 200\text{Mb}$
  - $np=100^2 \quad \text{size}(A) = 3.2\text{Gb}$
- ⇒ not a viable option in the end

## grid points setup

```
counter=0
do j=0,nely
  do i=0,nelx
    counter=counter+1
    x(counter)=dble(i)*Lx/dble(nelx)
    y(counter)=dble(j)*Ly/dble(nely)
  end do
end do
```

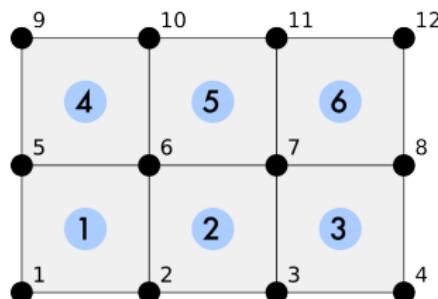


OUT/gridnodes.dat

xpos	ypos	node
0.00000	0.00000	1
1.00000	0.00000	2
2.00000	0.00000	3
3.00000	0.00000	4
0.00000	1.00000	5
1.00000	1.00000	6
2.00000	1.00000	7
3.00000	1.00000	8
0.00000	2.00000	9
1.00000	2.00000	10
2.00000	2.00000	11
3.00000	2.00000	12

# connectivity setup

```
counter=0
do j=1,nely
  do i=1,nelx
    counter=counter+1
    icon(1,counter)=i+(j-1)*(nelx+1)
    icon(2,counter)=i+1+(j-1)*(nelx+1)
    icon(3,counter)=i+j*(nelx+1)
    icon(4,counter)=i+j*(nelx+1)
  end do
end do
```



OUT/icon.dat

element # 1			
node 1:	1	at pos.	0.000
node 2:	2	at pos.	1.000
node 3:	6	at pos.	1.000
node 4:	5	at pos.	1.000
element # 2			
node 1:	2	at pos.	1.000
node 2:	3	at pos.	2.000
node 3:	7	at pos.	2.000
node 4:	6	at pos.	1.000
element # 3			
node 1:	3	at pos.	2.000
node 2:	4	at pos.	3.000
node 3:	8	at pos.	3.000
node 4:	7	at pos.	2.000
element # 4			
node 1:	5	at pos.	0.000
node 2:	6	at pos.	1.000
node 3:	10	at pos.	1.000
node 4:	9	at pos.	0.000

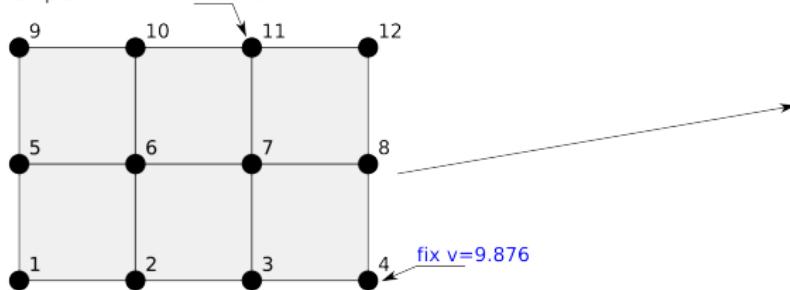
etc ...

## boundary conditions setup

```
bc_fix=.false.  
  
do i=1,np  
  if (x(i).lt.eps) then  
    bc_fix((i-1)*ndof+1)=.true. ; bc_val((i-1)*ndof+1)=0.d0  
    bc_fix((i-1)*ndof+2)=.true. ; bc_val((i-1)*ndof+2)=0.d0  
  endif  
  if (x(i).gt.(Lx-eps)) then  
    bc_fix((i-1)*ndof+1)=.true. ; bc_val((i-1)*ndof+1)=0.d0  
    bc_fix((i-1)*ndof+2)=.true. ; bc_val((i-1)*ndof+2)=0.d0  
  endif  
  if (y(i).lt.eps) then  
    bc_fix((i-1)*ndof+1)=.true. ; bc_val((i-1)*ndof+1)=0.d0  
    bc_fix((i-1)*ndof+2)=.true. ; bc_val((i-1)*ndof+2)=0.d0  
  endif  
  if (y(i).gt.(Ly-eps) ) then  
    bc_fix((i-1)*ndof+1)=.true. ; bc_val((i-1)*ndof+1)=0.d0  
    bc_fix((i-1)*ndof+2)=.true. ; bc_val((i-1)*ndof+2)=0.d0  
  endif  
end do
```

Example:

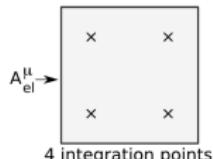
fix  $u=1.123$



	bc_fix	bc_val
u1	F	1
v1	F	2
u2	F	3
v2	F	4
u3	F	5
v3	F	6
u4	F	7
v4	T	9.876
u5	F	8
v5	F	9
u6	F	10
v6	F	11
u7	F	12
v7	F	13
u8	F	14
v8	F	15
u9	F	16
v9	F	17
u10	F	18
v10	F	19
u11	T	1.123
v11	F	21
u12	F	22
v12	F	23
		24

# Building the FEM matrix and rhs (1)

```
do iel=1,nel  
  
Ael=0.d0  
Bel=0.d0  
  
!==== integrate viscous term at 4 qpoints ====  
  
do iq=-1,1,2  
do jq=-1,1,2  
  
rq=iq/sqrt(3.d0) ; sq=jq/sqrt(3.d0) ; wq=1.d0  
  
! compute N(1),N(2),N(3),N(4)  
! compute dNdr(1),dNdr(2),dNdr(3),dNdr(4)  
! compute jcb(2x2) = jacobian J  
! compute jcob = |J|  
! compute jcobi(2x2) = inverse of jacobian  
! compute dNdx(1),dNdx(2),dNdx(3),dNdx(4)  
! build B(3x8) matrix  
  
Ael=Ael+matmul(transpose(Bmat),matmul(viscosity*Cmat,Bmat))*wq*jcob  
  
do i=1,m  
i1=2*i-1  
i2=2*i  
Bel(i1)=Bel(i1)+N(i)*jcob*wq*density*gx  
Bel(i2)=Bel(i2)+N(i)*jcob*wq*density*gy  
end do  
  
end do  
end do
```



$$A_{el} = \int_{\Omega_e} \mu \mathbf{B}^T \cdot \mathbf{C} \cdot \mathbf{B} d\Omega$$

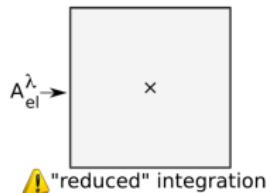
$$= \sum_{i=1}^n \sum_{j=1}^n w_i w_j |J| (\mu \mathbf{B}^T \cdot \mathbf{C} \cdot \mathbf{B})$$

$$B_{el} = \int_{\Omega_e} \mathbf{N}_b d\Omega$$

$$= \sum_{i=1}^n \sum_{j=1}^n w_i w_j |J| (\mathbf{N}_b)$$

## Building the FEM matrix and rhs (2)

```
!==== integrate penalty term at one qpoint ====
rq=0.d0 ; sq=0.d0 ; wq=4.d0
! compute N(1),N(2),N(3),N(4)
! compute dNdr(1),dNdr(2),dNdr(3),dNdr(4)
! compute jcb(2x2) = jacobian J
! compute jacob = |J|
! compute jcobi(2x2) = inverse of jacobian
! compute dNdx(1),dNdx(2),dNdx(3),dNdx(4)
! build B(3x8) matrix
Ael=Ael+matmul(transpose(Bmat),matmul(penalty*Kmat,Bmat))*wq*jcob
!==== assemble ====
[...]
end do
```

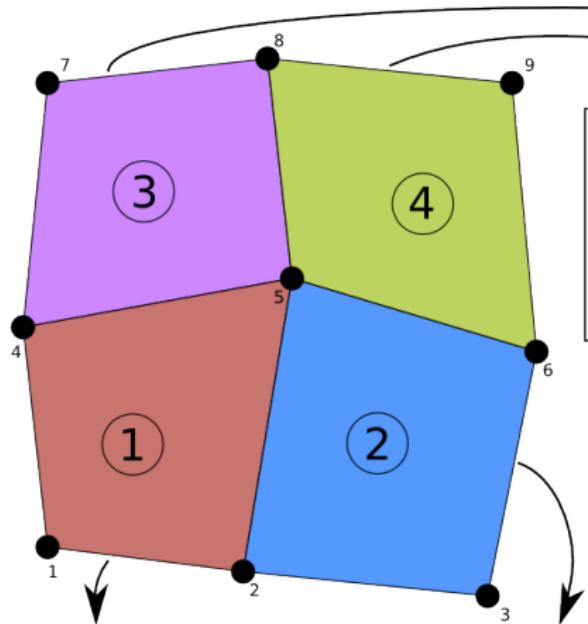


⚠ "reduced" integration

$$A_{el} = \int_{\Omega_e} \lambda \mathbf{B}^T \cdot \mathbf{K} \cdot \mathbf{B} d\Omega$$

$$= \sum_{i=1}^n \sum_{j=1}^n w_i w_j |J| (\lambda \mathbf{B}^T \cdot \mathbf{K} \cdot \mathbf{B})$$

# assembly (1)



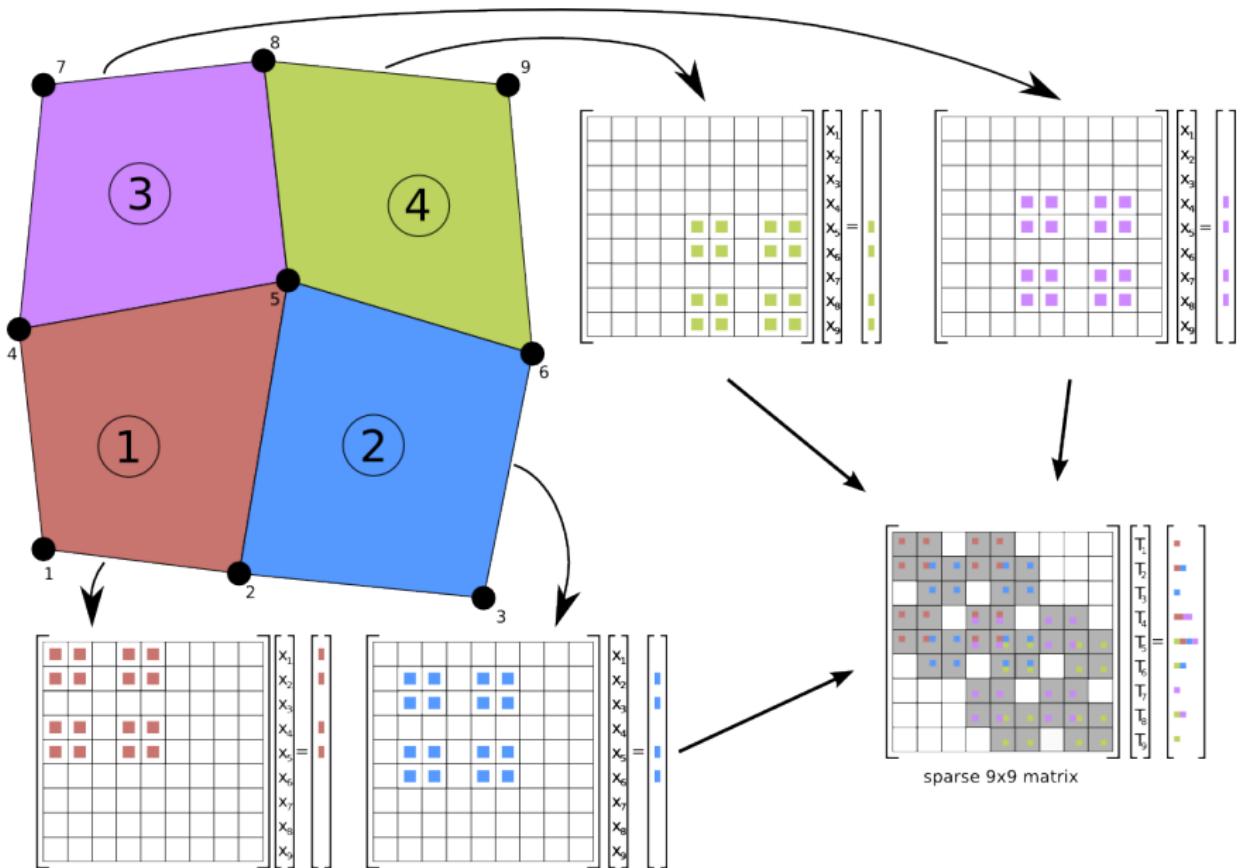
$$\begin{bmatrix} & & \\ & & \\ & & \end{bmatrix} \quad \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \\ x_5 \\ x_6 \\ x_7 \\ x_8 \\ x_9 \end{bmatrix} = \begin{bmatrix} & & \\ & & \\ & & \end{bmatrix}$$

$$\begin{bmatrix} & & \\ & & \\ & & \end{bmatrix} \quad \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \\ x_5 \\ x_6 \\ x_7 \\ x_8 \\ x_9 \end{bmatrix} = \begin{bmatrix} & & \\ & & \\ & & \end{bmatrix}$$

$$\begin{bmatrix} & & \\ & & \\ & & \end{bmatrix} \quad \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \\ x_5 \\ x_6 \\ x_7 \\ x_8 \\ x_9 \end{bmatrix} = \begin{bmatrix} & & \\ & & \\ & & \end{bmatrix}$$

$$\begin{bmatrix} & & \\ & & \\ & & \end{bmatrix} \quad \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \\ x_5 \\ x_6 \\ x_7 \\ x_8 \\ x_9 \end{bmatrix} = \begin{bmatrix} & & \\ & & \\ & & \end{bmatrix}$$

## assembly (2)



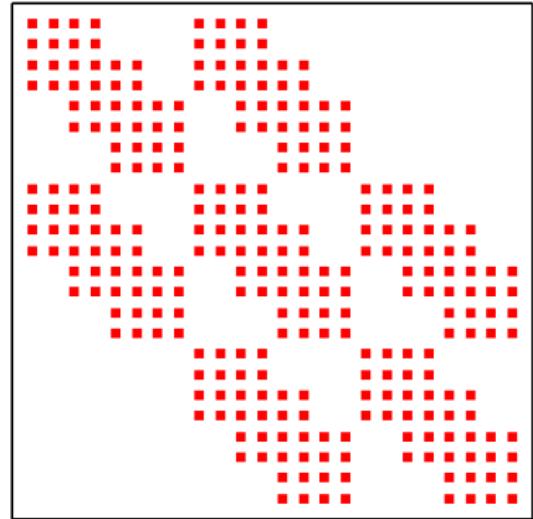
## assembly (3)

```
do k1=1,m
  ik=icon(k1,iel)
  do i1=1,ndof
    ikk=ndof*(k1-1)+i1
    m1=ndof*(ik-1)+i1
    do k2=1,m
      jk=icon(k2,iel)
      do i2=1,ndof
        jkk=ndof*(k2-1)+i2
        m2=ndof*(jk-1)+i2
        A(m1,m2)=A(m1,m2)+Ael(ikk,jkk)
        C(m1,m2)=.true.
      end do
    end do
    B(m1)=B(m1)+Bel(ikk)
  end do
end do
```

(ikk,jkk) : coordinates in local matrix

(m1,m2) : coordinates in global matrix

FEM matrix sparsity pattern :



$$N_{fem} = ndof \times np = 2 \times 4 \times 3 = 24$$

## imposing boundary conditions (1)

Let us consider an algebraic system of the form  $\mathbf{A.x} = \mathbf{B}$  :

$$\begin{pmatrix} A_{11} & A_{12} & A_{13} & A_{14} \\ A_{21} & A_{22} & A_{23} & A_{24} \\ A_{31} & A_{32} & A_{33} & A_{34} \\ A_{41} & A_{42} & A_{43} & A_{44} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{pmatrix}$$

Let us assume that we want to impose  $x_3 = x^{bc}$  on the third node. Then system writes

## imposing boundary conditions (1)

Let us consider an algebraic system of the form  $\mathbf{A} \cdot \mathbf{x} = \mathbf{B}$  :

$$\begin{pmatrix} A_{11} & A_{12} & A_{13} & A_{14} \\ A_{21} & A_{22} & A_{23} & A_{44} \\ A_{31} & A_{32} & A_{33} & A_{44} \\ A_{41} & A_{42} & A_{43} & A_{44} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{pmatrix}$$

Let us assume that we want to impose  $x_3 = x^{bc}$  on the third node. Then system writes

$$\begin{pmatrix} A_{11} & A_{12} & A_{13} & A_{14} \\ A_{21} & A_{22} & A_{23} & A_{44} \\ 0 & 0 & 1 & 0 \\ A_{41} & A_{42} & A_{43} & A_{44} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 \\ B_2 \\ x^{bc} \\ B_4 \end{pmatrix}$$

## imposing boundary conditions (1)

Let us consider an algebraic system of the form  $\mathbf{A} \cdot \mathbf{x} = \mathbf{B}$  :

$$\begin{pmatrix} A_{11} & A_{12} & A_{13} & A_{14} \\ A_{21} & A_{22} & A_{23} & A_{44} \\ A_{31} & A_{32} & A_{33} & A_{44} \\ A_{41} & A_{42} & A_{43} & A_{44} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{pmatrix}$$

Let us assume that we want to impose  $x_3 = x^{bc}$  on the third node. Then system writes

$$\begin{pmatrix} A_{11} & A_{12} & 0 & A_{13} \\ A_{21} & A_{22} & 0 & A_{23} \\ 0 & 0 & 1 & 0 \\ A_{41} & A_{42} & 0 & A_{43} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 - A_{13}x^{bc} \\ B_2 - A_{23}x^{bc} \\ x^{bc} \\ B_4 - A_{43}x^{bc} \end{pmatrix}$$

## imposing boundary conditions (1)

Let us consider an algebraic system of the form  $\mathbf{A} \cdot \mathbf{x} = \mathbf{B}$  :

$$\begin{pmatrix} A_{11} & A_{12} & A_{13} & A_{14} \\ A_{21} & A_{22} & A_{23} & A_{44} \\ A_{31} & A_{32} & A_{33} & A_{44} \\ A_{41} & A_{42} & A_{43} & A_{44} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 \\ B_2 \\ B_3 \\ B_4 \end{pmatrix}$$

Let us assume that we want to impose  $x_3 = x^{bc}$  on the third node. Then system writes

$$\begin{pmatrix} A_{11} & A_{12} & 0 & A_{13} \\ A_{21} & A_{22} & 0 & A_{23} \\ 0 & 0 & A_{33} & 0 \\ A_{41} & A_{42} & 0 & A_{43} \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} B_1 - A_{13}x^{bc} \\ B_2 - A_{23}x^{bc} \\ A_{33} * x^{bc} \\ B_4 - A_{43}x^{bc} \end{pmatrix}$$

## imposing boundary conditions (2)

```
do i=1,Nfe}m
  if (bc_fix(i)) then
    Aref=A(i,i)
    do j=1,Nfem
      B(j)=B(j)-A(i,j)*bc_val(i)
      A(i,j)=0.d0
      A(j,i)=0.d0
    end do
    A(i,i)=Aref
    B(i)=Aref*bc_val(i)
  end if
end do
```

## Solving the system

⇒ [www.netlib.org](http://www.netlib.org)

- ▶ dgeco factors a double precision matrix by gaussian elimination and estimates the condition of the matrix.
- ▶ dgesl solves the double precision system using the factors computed by dgeco or dgfa.

```
job=0
allocate(work(Nfem))
allocate(ipvt(Nfem))
call DGECO (A, Nfem, Nfem, ipvt, rcond, work)
call DGESL (A, Nfem, Nfem, ipvt, B, job)
deallocate(ipvt)
deallocate(work)
```

The solution is stored in B, we must now split it into u and v arrays :

```
do i=1,np
  u(i)=B((i-1)*ndof+1)
  v(i)=B((i-1)*ndof+2)
end do
```

## Compute pressure (1)

```
do iel=1,nel
[ compute N(1:4),dNdx(1:4),dNdy(1:4)
at center of element]

xq=0.d0
yq=0.d0
exxq=0.d0
eyyq=0.d0
do k=1,m
  xq=xq+N(k)*x(icon(k,iel))
  yq=yq+N(k)*y(icon(k,iel))
  exxq=exxq+ dNdx(k)*u(icon(k,iel))
  eyyq=eyyq+ dNdy(k)*v(icon(k,iel))
end do

press(iel)=-penalty*(exxq+eyyq)

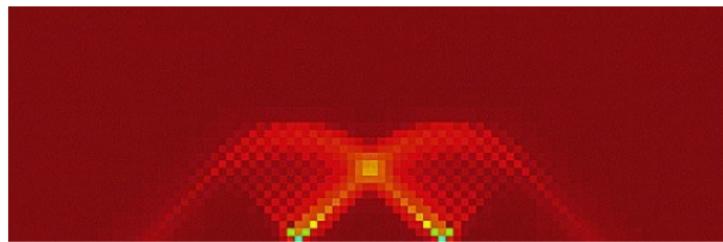
end do
```

$$\begin{aligned} p &= -\lambda \nabla \cdot \mathbf{v} \\ &= -\lambda (\dot{\epsilon}_{xx} + \dot{\epsilon}_{yy}) \end{aligned}$$

## Compute pressure (2)



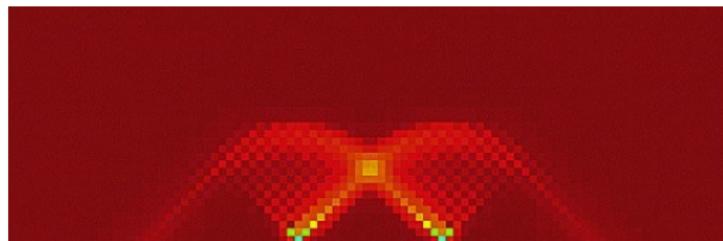
pressure field likely to display checkerboard pattern :



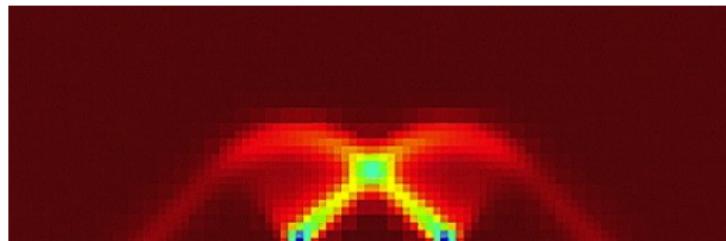
## Compute pressure (2)



pressure field likely to display checkerboard pattern :



↓ Smoothing/Filtering



Thieulot et al, JGR, 2008

# Compiling and running the code

## Compilation

```
> gfortran blas_routines.f90 linpack_d.f90 simplefem.f90 -o simplefem
```

## Run

```
> ./simplefem
```

## Produce figures :

```
> cd OUT  
> gnuplot script
```

## Analytical stokes benchmark

- ▶ square domain  $\Omega = [0, 1] \times [0, 1]$ ,
- ▶ boundary conditions  $\mathbf{v} = \mathbf{0}$  on  $\Gamma$
- ▶ viscosity  $\mu=1$
- ▶ body force

$$\begin{aligned} b_x &= (12 - 24y)x^4 + (-24 + 48y)x^3 + (-48y + 72y^2 - 48y^3 + 12)x^2 \\ &\quad + (-2 + 24y - 72y^2 + 48y^3)x + 1 - 4y + 12y^2 - 8y^3 \\ b_y &= (8 - 48y + 48y^2)x^3 + (-12 + 72y - 72y^2)x^2 \\ &\quad + (4 - 24y + 48y^2 - 48y^3 + 24y^4)x - 12y^2 + 24y^3 - 12y^4 \end{aligned}$$

Exact solution is

$$\begin{aligned} u(x, y) &= x^2(1-x)^2(2y - 6y^2 + 4y^3) \\ v(x, y) &= -y^2(1-y)^2(2x - 6x^2 + 4x^3) \\ p(x, y) &= x(1-x) \end{aligned}$$

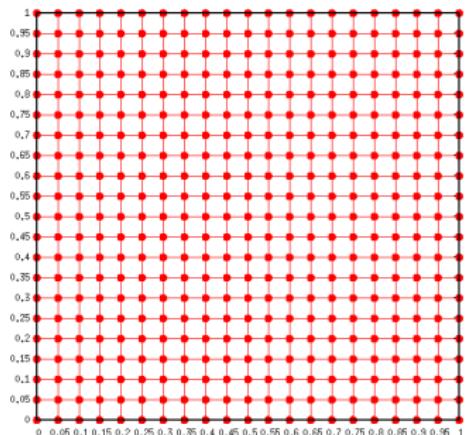
## Benchmark setup

$L_x=1$   
 $L_y=1$

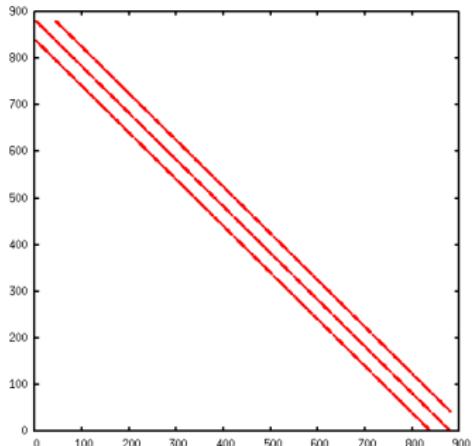
$n_{nx}=21$   
 $n_{ny}=21$   
 $np=n_{nx}*n_{ny} = 441$

$n_{elx}=n_{nx}-1=20$   
 $n_{ely}=n_{ny}-1=20$   
 $n_{el}=n_{elx}*n_{ely}=400$

$Nfem=ndof*np=882$

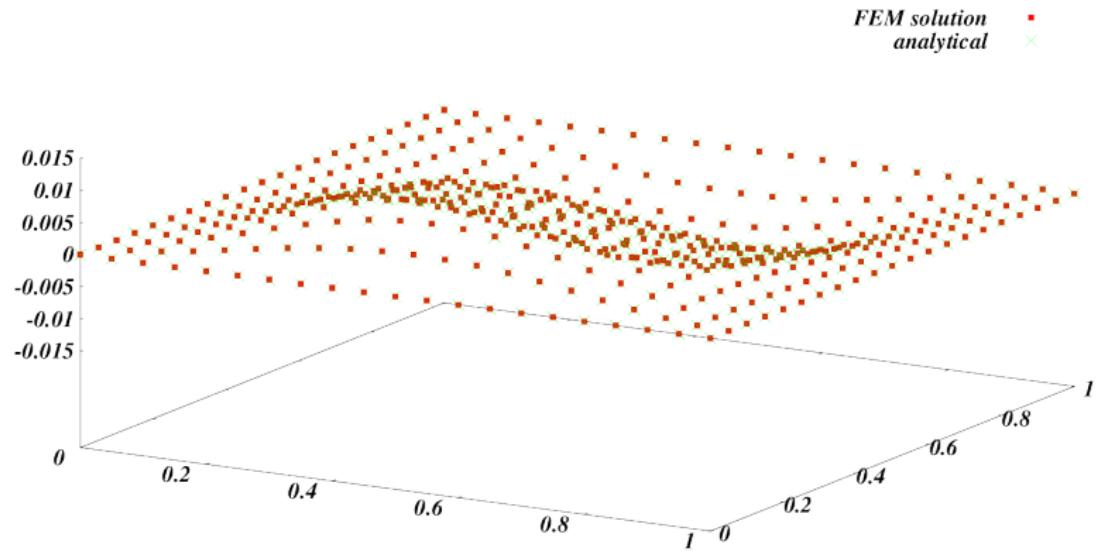


grid

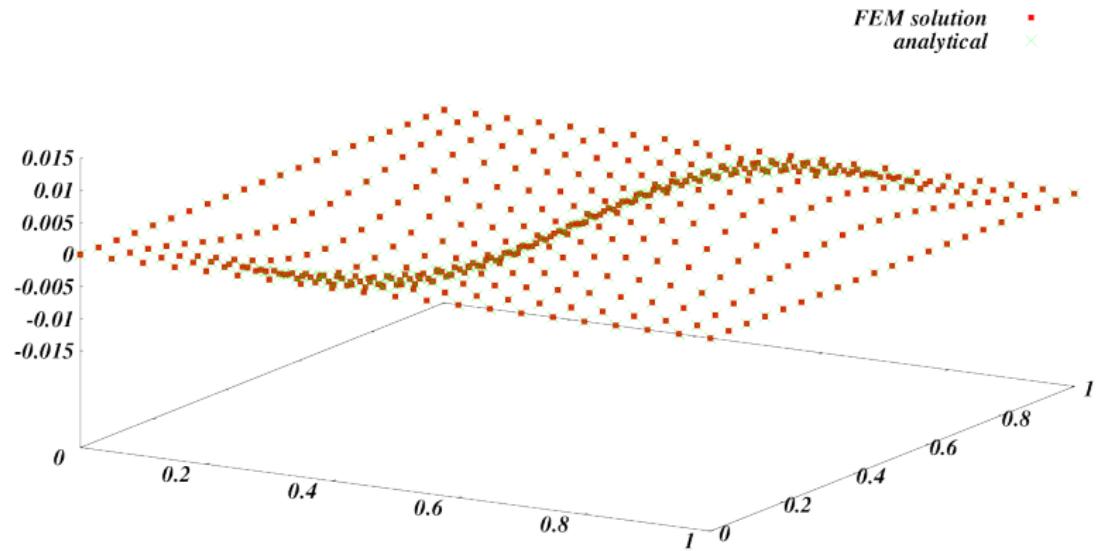


matrix

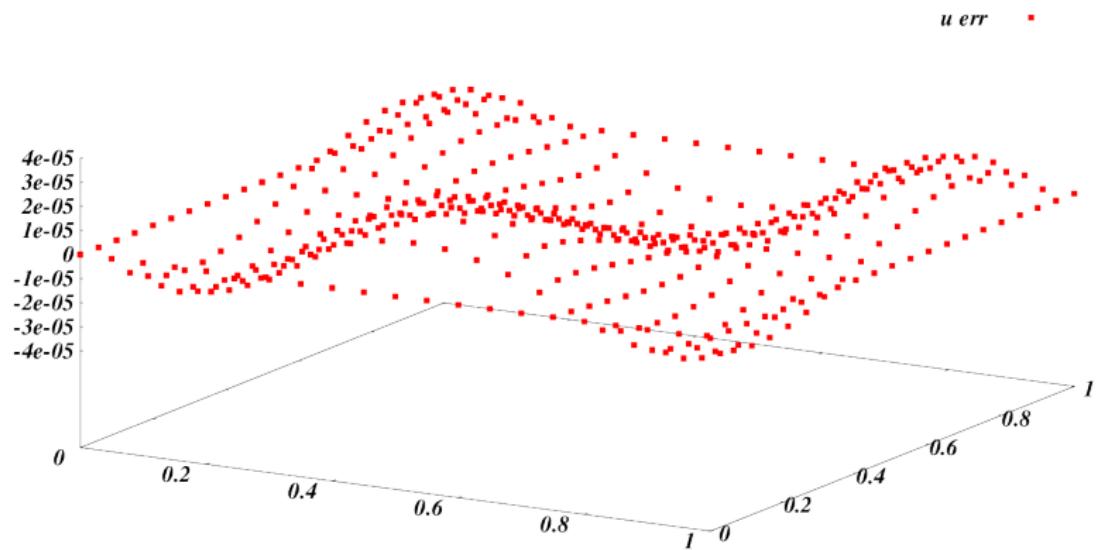
## Results (1) : u field



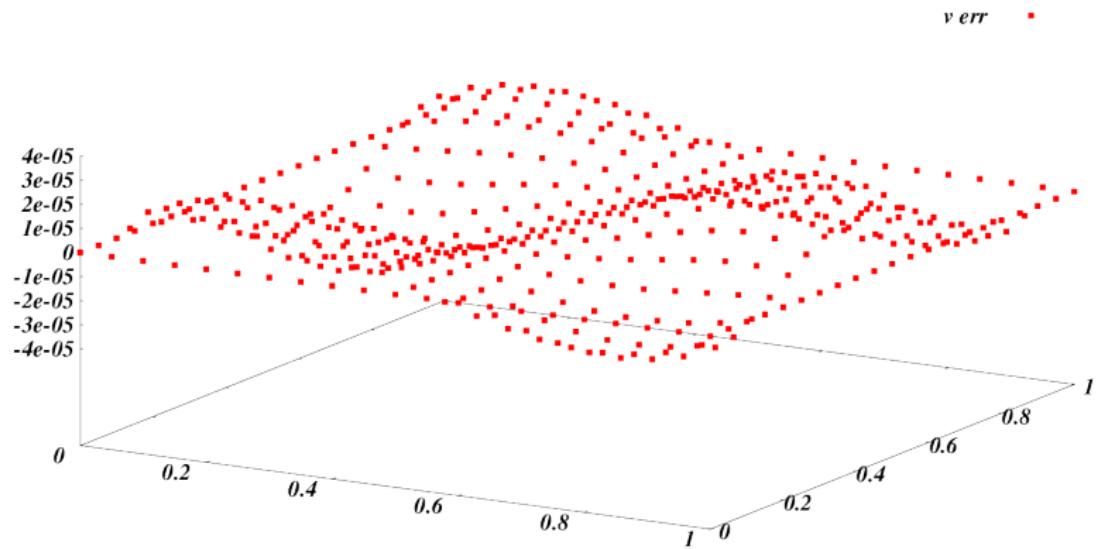
## Results (2) : v field



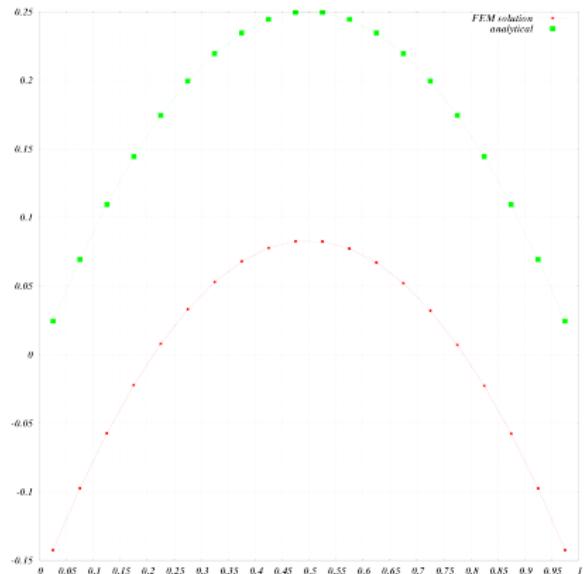
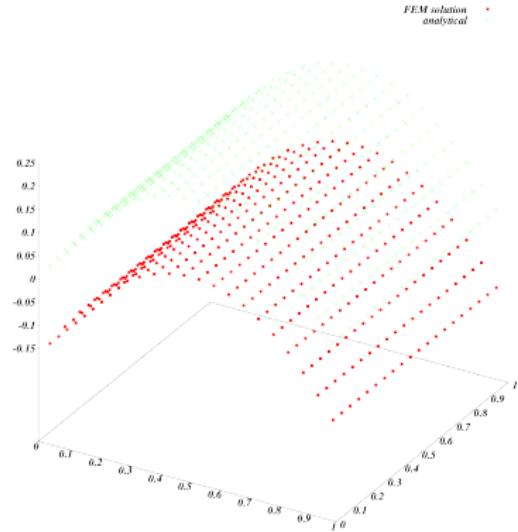
## Results (3) : Velocity field error ( $u - u_{th}$ )



## Results (4) : Velocity field error ( $v - v_{th}$ )

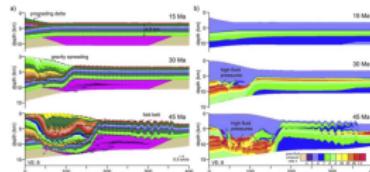


## Results (5) : p field



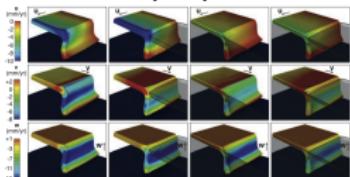
# Applications to geodynamics

## ► SOPALE (2D)



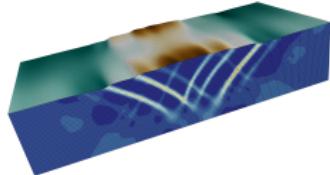
<http://geodynamics.oceanography.dal.ca/sopaledoc.html>

## ► DOUAR (3D)



<http://www.cedrichieulot.net/douar.html>

## ► FANTOM (2D,3D)



<http://www.cedrichieulot.net/fantom.html>

## homework

- ▶ optimise
- ▶ try other benchmark
- ▶ extend to 3D
- ▶ implement non-penalised (i.e. mixed) formulations
- ▶ implement other element
- ▶ compile with optimised BLAS library of your computer OS
- ▶ implement clever storage of matrix terms (CSR/CSC)  
→ change solver to Pardiso or MUMPS
- ▶ implement particle-in-cell technique